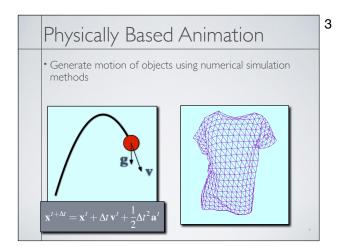
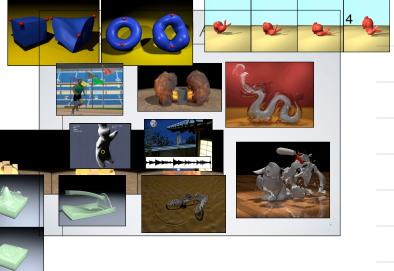
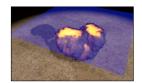
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CS-184: Computer Graphics	
Lecture #18: Simulation Basics	
Prof. James O'Brien University of California, Berkeley	
(With some slides from Prof. Ren Ng who is really an awesome guy.)	
	1.
Today	2
Introduction to Simulation Basic particle systems	
Time integration (simple version)	

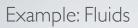


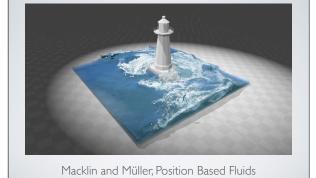


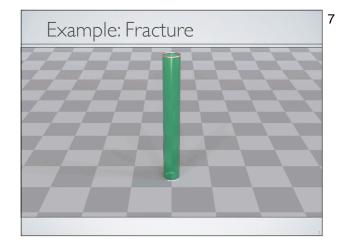






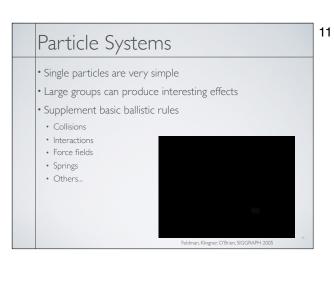








9 Particle Systems • Single particles are very simple • Large groups can produce interesting effects Supplement basic ballistic rules Collisions Interactions Force fields Springs Others... 10 PARTICLE DREAMS Karl Sims Optomystic



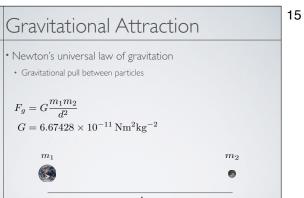
Basic Particles

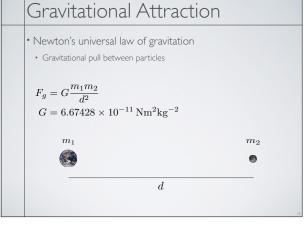
- Basic governing equation
- $\cdot f$ is a sum of a number of things
- Gravity: constant downward force proportional to mass
- Simple drag: force proportional to negative velocity
- Particle interactions: particles mutually attract and/or repell
- Beware $O(n^2)$ complexity!
- Force fields
- Wind forces
- User interaction

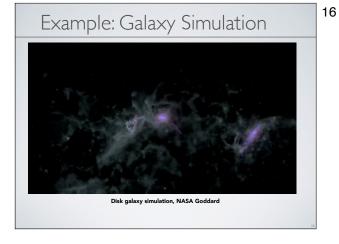
 $\ddot{x} = \frac{1}{m}f$

12			

13 Basic Particles • Properties other than position • Color • Temp • Age Differential equations also needed to govern these properties Collisions and other constrains directly modify position and/or velocity 14 Particle Rules Multiple Burst Bryan E. Feldman, James F. O'Brien, and Okan Arikan. "Animating Suspended Particle Explosions". In Proceedings of ACM SIGGRAPH 2003, pages 708–715, August 2003.







- Euler's Method
- Simple
- Commonly used
- Very inaccurate
- Most often goes unstable

$$\mathbf{x}^{t+\Delta t} = \mathbf{x}^t + \Delta t \, \mathbf{\dot{x}}^t$$

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$$\mathbf{\dot{x}}^{t+\Delta t} = \mathbf{\dot{x}}^t + \Delta t \, \mathbf{\ddot{x}}^t$$

Integration

• For now let's pretend

$$\boldsymbol{f} = m\boldsymbol{v}$$

• Velocity (rather than acceleration) is a function of force

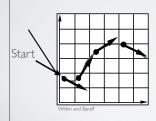


 $\dot{\boldsymbol{x}} = f(\boldsymbol{x}, t)$

Note: Second order ODEs can be turned into first order ODEs using extra variables.

$$\boldsymbol{f} = m \boldsymbol{v}$$

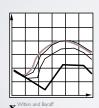
• Velocity (rather than acceleration) is a function of force



$$\dot{\boldsymbol{x}} = \mathsf{f}(\boldsymbol{x},t)$$

Integration

- With numerical integration, errors accumulate
- Euler integration is particularly bad



$$x := x + \Delta t \ \mathsf{f}(\boldsymbol{x}, t)$$

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- Stability issues can also arise
- Occurs when errors lead to larger errors
- Often more serious than error issues





 $\dot{\boldsymbol{x}} = [-\sin(\omega t) , -\cos(\omega t)]$

Integration

Modified Euler

$$\boldsymbol{x}^{t+\Delta t} = \boldsymbol{x}^t + \frac{\Delta t}{2} \left(\dot{\boldsymbol{x}}^t + \dot{\boldsymbol{x}}^{t+\Delta t} \right)$$

$$\dot{\boldsymbol{x}}^{t+\Delta t} = \dot{\boldsymbol{x}}^t + \Delta t \ \ddot{\boldsymbol{x}}^t$$

$$oldsymbol{x}^{t+\Delta t} = oldsymbol{x}^t + \Delta t \ \dot{oldsymbol{x}}^t + rac{(\Delta t)^2}{2} \ \ddot{oldsymbol{x}}^t$$

- Midpoint method
- a. Compute half Euler step
- b. Eval. derivative at halfway
- c. Retake step
- Other methods
- Verlet
- Runge-Kutta
- And many others...



X

23

Δ

Integration

- Implicit methods
- Informally (incorrectly) called backward methods
- Use derivatives in the future for the current step

$$\begin{split} \boldsymbol{x}^{t+\Delta t} &= \boldsymbol{x}^t + \Delta t \; \dot{\boldsymbol{x}}^{t+\Delta t} \\ \dot{\boldsymbol{x}}^{t+\Delta t} &= \dot{\boldsymbol{x}}^t + \Delta t \; \ddot{\boldsymbol{x}}^{t+\Delta t} \\ \\ \dot{\boldsymbol{x}}^{t+\Delta t} &= \mathsf{V}(\boldsymbol{x}^{t+\Delta t}, \dot{\boldsymbol{x}}^{t+\Delta t}, t + \Delta t) \\ \\ \ddot{\boldsymbol{x}}^{t+\Delta t} &= \mathsf{A}(\boldsymbol{x}^{t+\Delta t}, \dot{\boldsymbol{x}}^{t+\Delta t}, t + \Delta t) \end{split}$$

- Implicit methods
- Informally (incorrectly) called backward methods
- Use derivatives in the future for the current step

$$\begin{split} \dot{\boldsymbol{x}}^{t+\Delta t} &= \dot{\boldsymbol{x}}^t + \Delta t \; \mathsf{V}(\boldsymbol{x}^{t+\Delta t}, \dot{\boldsymbol{x}}^{t+\Delta t}, t + \Delta t) \\ \dot{\boldsymbol{x}}^{t+\Delta t} &= \dot{\boldsymbol{x}}^t + \Delta t \; \mathsf{A}(\boldsymbol{x}^{t+\Delta t}, \dot{\boldsymbol{x}}^{t+\Delta t}, t + \Delta t) \end{split}$$

- Solve nonlinear problem for $oldsymbol{x}^{t+\Delta t}$ and $\dot{oldsymbol{x}}^{t+\Delta t}$
- This is fully implicit backward Euler
- Many other implicit methods exist...
- Modified Euler is *partially* implicit as is Verlet

Temp Slide

Need to draw reverse diagrams....

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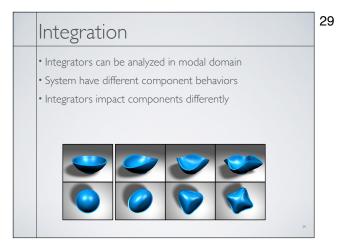
$$\mathsf{V}(\boldsymbol{x}^{t+\Delta t}, \dot{\boldsymbol{x}}^{t+\Delta t}) \approx \mathsf{V}(\boldsymbol{x}^t, \dot{\boldsymbol{x}}^t) + \mathbf{A} \cdot (\Delta \boldsymbol{x}) + \mathbf{B} \cdot (\Delta \dot{\boldsymbol{x}})$$

$$\mathsf{A}(\boldsymbol{x}^{t+\Delta t}, \dot{\boldsymbol{x}}^{t+\Delta t}) \approx \mathsf{A}(\boldsymbol{x}^t, \dot{\boldsymbol{x}}^t) + \mathbf{C} \cdot (\Delta \boldsymbol{x}) + \mathbf{D} \cdot (\Delta \dot{\boldsymbol{x}})$$

$$\begin{bmatrix} \boldsymbol{x}^{t+\Delta t} \\ \dot{\boldsymbol{x}}^{t+\Delta t} \end{bmatrix} = \begin{bmatrix} \boldsymbol{x}^t \\ \dot{\boldsymbol{x}}^t \end{bmatrix} + \Delta t \left(\begin{bmatrix} \dot{\boldsymbol{x}}^t \\ \ddot{\boldsymbol{x}}^t \end{bmatrix} + \begin{bmatrix} \mathbf{A} \ \mathbf{B} \\ \mathbf{C} \ \mathbf{D} \end{bmatrix} \begin{bmatrix} \Delta \boldsymbol{x} \\ \Delta \dot{\boldsymbol{x}} \end{bmatrix} \right)$$

- Explicit methods can be conditionally stable
- Depends on time-step and stiffness of system
- Fully implicit can be **un**conditionally stable
- May still have large errors
- Semi-implicit can be conditionally stable
- Nonlinearities can cause instability
- Generally more stable than explicit
- Comparable errors as explicit
- Often show up as excessive damping

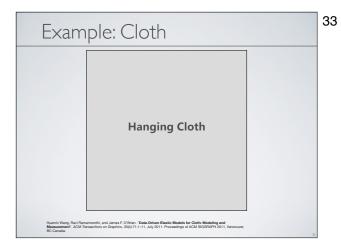
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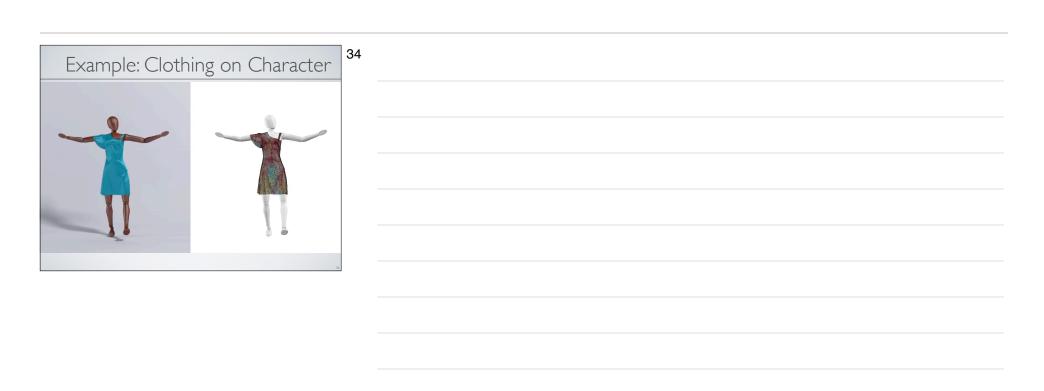


Example: Mass Spring Rope



Example: Hair Slide from Ren Ng





A Simple Spring

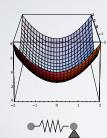
• Ideal **zero**-length spring

$$\textbf{\textit{f}}_{a \rightarrow b} = k_{s}(\textbf{\textit{b}} - \textbf{\textit{a}})$$

- ullet Force pulls points together $oldsymbol{f}_{b
 ightarrow a}$ = $-oldsymbol{f}_{a
 ightarrow b}$
- Strength proportional to distance

A Simple Spring

• Energy potential



$$E = 1/2 k_S(\boldsymbol{b} - \boldsymbol{a}) \cdot (\boldsymbol{b} - \boldsymbol{a})$$

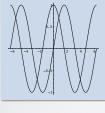
$$egin{aligned} oldsymbol{f}_{a
ightarrow b} &= k_{S}(oldsymbol{b} - oldsymbol{a}) \ oldsymbol{f}_{b
ightarrow a} &= -oldsymbol{f}_{a
ightarrow b} \end{aligned}$$

$$f_{b \to a} = -f_{a \to b}$$

$$m{f}_a = -
abla_a E = -\left[rac{\partial E}{\partial a_x}, rac{\partial E}{\partial a_y}, rac{\partial E}{\partial a_z}
ight]$$

A Simple Spring

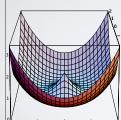
• Energy potential: kinetic **vs** elastic



$$E = 1/2 k_{\mathcal{S}}(\boldsymbol{b} - \boldsymbol{a}) \cdot (\boldsymbol{b} - \boldsymbol{a})$$

$$E = 1/2 \ m(\dot{\boldsymbol{b}} - \dot{\boldsymbol{a}}) \cdot (\dot{\boldsymbol{b}} - \dot{\boldsymbol{a}})$$





 $E = k_s \left(||\boldsymbol{b} - \boldsymbol{a}|| - l \right)^2$

Comments on Springs

- Springs with zero rest length are linear
- Springs with non-zero rest length are nonliner
- Force *magnitude* linear w/ discplacement (from rest length)
- Force direction is non-linear
- Singularity at

$$||\boldsymbol{b} - \boldsymbol{a}|| = 0$$

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• "Mass proportional" damping

$$f = -k_d \dot{a}$$

- Behaves like viscous drag on all motion
- · Consider a pair of masses connected by a spring
- How to model rusty vs oiled spring
- Should internal damping slow group motion of the pair?
- Can help stability... up to a point

Damping

• "Stiffness proportional" damping

$$f_a = -k_d \frac{\boldsymbol{b} - \boldsymbol{a}}{||\boldsymbol{b} - \boldsymbol{a}||^2} (\boldsymbol{b} - \boldsymbol{a}) \cdot (\dot{\boldsymbol{b}} - \dot{\boldsymbol{a}})$$

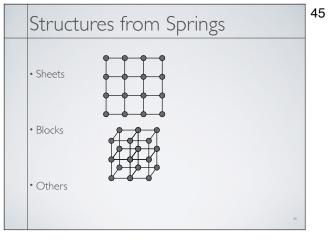
- Behaves viscous drag on change in spring length
- Consider a pair of masses connected by a spring
- How to model rusty vs oiled spring
- Should internal damping slow group motion of the pair?

Spring Constants

- Constant $k_{\mathcal{S}}$ gives inconsistent results with different discretizations
- Change in length is not what we want to measure
- Strain: change in length as fraction of original length

$$\epsilon = \frac{\Delta l}{l_0} \quad \text{Nice and simple for ID...}$$

- 4	



Structures from Springs

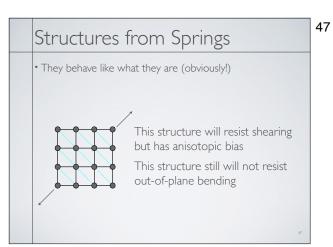
• They behave like what they are (obviously!)

shearing

This structure will not resist

This structure will not resist outof-plane bending either...







Structures from Springs

• They behave like what they are (obviously!)



This structure will resist shearing Less bias

Interference between spring sets

This structure still will not resist out-of-plane bending

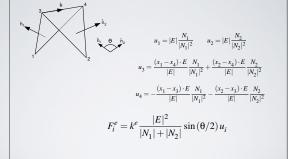


This structure will resist shearing Less bias Interference between spring sets

This structure will resist out-ofplane bending Interference between spring sets Odd behavior

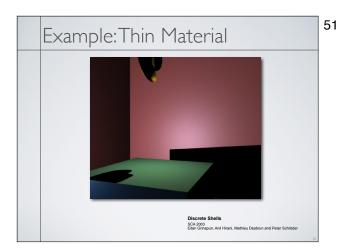
How do we set spring constants?

Edge Springs

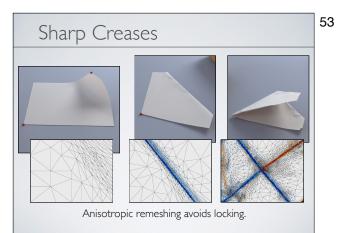


From Bridson et al., 2003, also see Grinspun et al., 2003

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Fracture Dominated by discretization artifacts Fracture Natural appearance

